

4.0 TANK DESIGN

4.1 Lower Cone Thickness

Lower Cone Design: Ref [4]

Assumptions: 1) The axial force due to the weight of the contents bearing on the laminate below the point of interest acts in a vertical direction (worst case) for the top and bottom points of the cone. The mid point sees an axial force parallel to the angle of the cone.

Knowns: Stitchmat Laminate

Upper Diameter, $D_1 = 132$ in

Middle Diameter, $D_M = 69$ in

Lower Diameter, $D_2 = 6$ in

Cone Height, $H = 88.25$ in

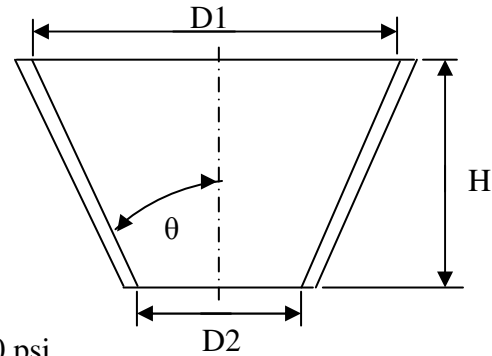
Fluid Head above Cone, $H_h = 258$ in

Cone Angle, $\theta = 35.5$ deg

Density of Water, $\gamma = 62.4$ lbs/ft³

Additional Pressure, $P_{ADD} = 0$

Maximum Stress (Hoop/Axial), $\sigma_{max} = 2250$ psi



Let the initial estimated thickness, t_{est} , be 0.605 in. for the top half of cone and 0.370 in. for the bottom half of cone.

Determine hydrostatic pressures at top, middle and bottom of cone,

- Hydrostatic Pressure at Top, $P_{top} = \frac{\gamma}{12^3} H_h SG + P_{ADD} = 16.77$ psi
- Hydrostatic Pressure at Middle, $P_{mid} = \frac{\gamma}{12^3} \left(H_h + \frac{H}{2} \right) SG + P_{ADD} = 19.64$ psi
- Hydrostatic Pressure at Bottom, $P_{bot} = \frac{\gamma}{12^3} (H_h + H) SG + P_{ADD} = 22.51$ psi

Determine the hoop and axial stresses due to the hydrostatic pressure [4],

- at Top,

$$\sigma_{top,axial} = \frac{P_{top} R_1}{2t_{est,top} \cos \theta} = 1124$$
 psi

$$\sigma_{top,hoop} = \frac{P_{top} R_1}{t_{est,top} \cos \theta} = 2248$$
 psi

- at Middle,

$$\sigma_{mid,axial} = \frac{P_{mid} R_M}{2t_{est,bot} \cos \theta} = 1125$$
 psi

$$\sigma_{mid,hoop} = \frac{P_{mid} R_M}{t_{est,bot} \cos \theta} = 2250$$
 psi